

A woman with long dark hair, wearing a dark top, is looking into a large, open industrial machine. The machine has a complex internal structure with various components, including what appears to be a large cylindrical chamber and several smaller circular openings. The machine is made of metal and has a clean, professional appearance. The background is a plain, light-colored wall.

# Physics of turbomolecular pumps and demonstration of conductance influence in HV

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# Agenda



1	Basics of turbomolecular pumps
2	Key parameters of turbomolecular pumps
3	The operational diagram: The turbo's "ID card"
4	Conductance influence in HV
5	Summary

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# Basics of turbomolecular pumps



# Basics of turbomolecular pumps



## Turbomolecular Pump

- Kinetic vacuum pump
- High vacuum pump w/ typical operating pressures between  $10^{-3}$  and  $10^{-11}$  mbar
- Cannot compress against atmospheric pressure, i.e. a “backing pump” to further compress the gas to 1000 mbar
- Typical pumping speed sizes between 10 l/s and ~ 4000 l/s
- Main pumping principle: fast spinning rotor transfers momentum to gas particles in a molecular flow regime
- Typical rotor speeds: 500 – 1500 Hz (24000 – 90000 rpm)
- Technical challenges: rotor design, bearing concept (mechanical, magnetic, hybrid), safety

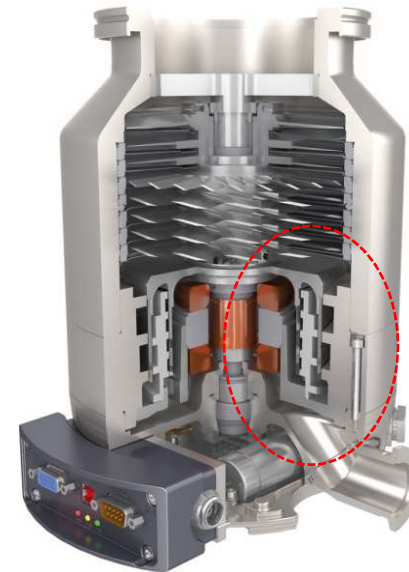
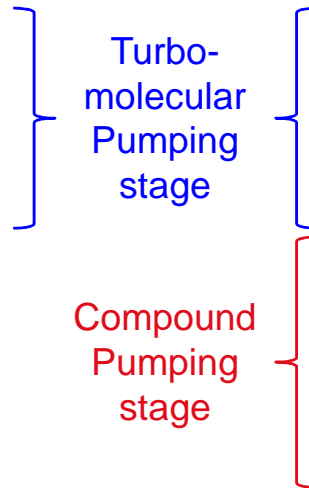


# Basics of turbomolecular pumps



## “Classic” turbo pumps and “Wide-range” turbo pumps

“Classic” Turbomolecular Pump      “Wide Range”



Pump containing only a **Turbomolecular pumping stage**

Pump containing a **Turbomolecular pumping stage & Compound pumping stage**

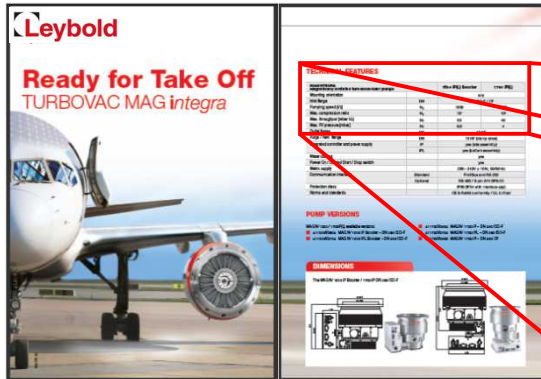
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# Key parameters of turbomolecular pumps



MAGiTEGRA Magnetically levitated turbomolecular pumps		1600 iP(L) Booster	1700 iP(L)
Mounting orientation		any	
Inlet flange	DN	250 ISO-F / CF	
Pumping speed [l/s] <b>1</b>	N <sub>2</sub>	1600	1650
Max. compression ratio <b>2</b>	N <sub>2</sub>	10 <sup>7</sup>	10 <sup>8</sup>
Max. throughput [mbar l/s] <b>3</b>	N <sub>2</sub>	60	40
Max. FV pressure [mbar] <b>4</b>	N <sub>2</sub>	0,9	4

## Key Performance Indicators

- 1 Pumping Speed** Maximum pumping speed at low backing pressures
- 2 Maximum Compression Ratio** Maximum compression (Inlet pressure/Outlet pressure) under no flow conditions (zero pumping speed).
- 3 Maximum Throughput** Maximum gas flow before the pumping speed starts to degrade by more than 10%
- 4 Maximum Fore Vacuum (FV) Pressure** Maximum exhaust pressure before pumping speed starts to degrade by more than 10%.



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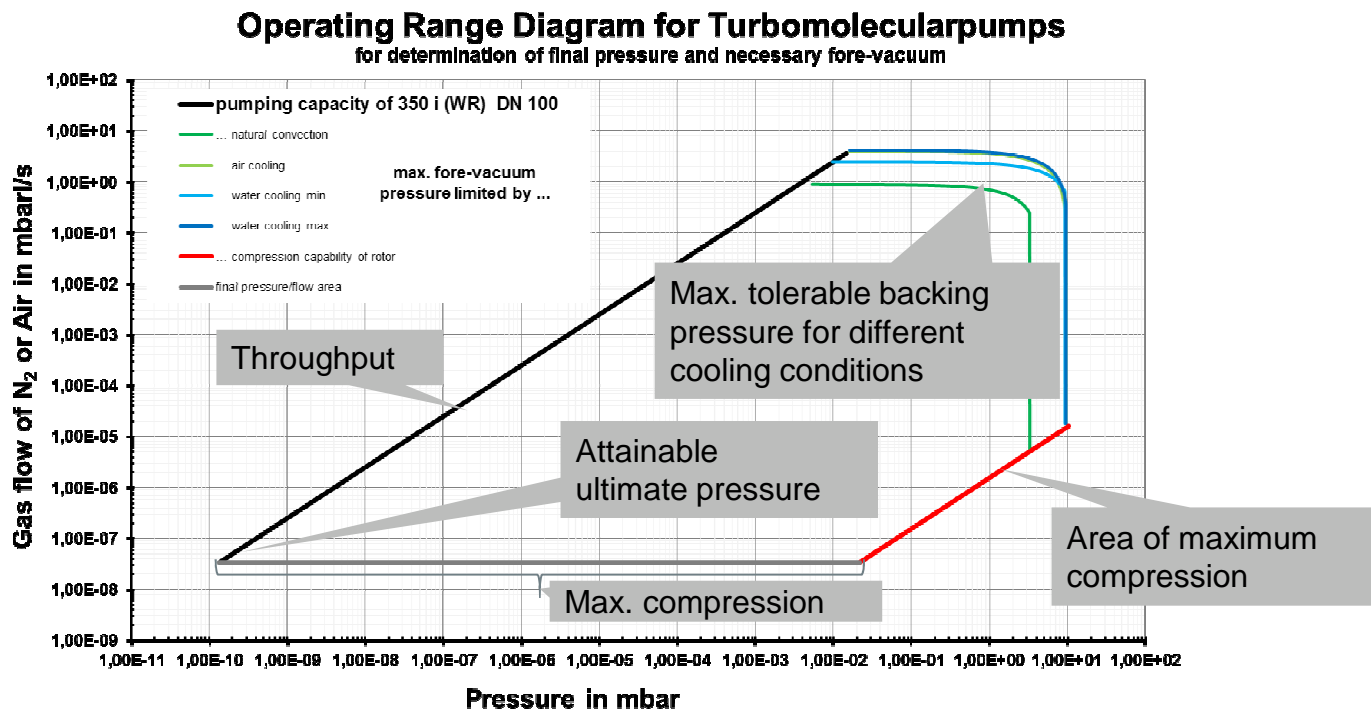


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# The operational diagram: The turbo's "ID card"



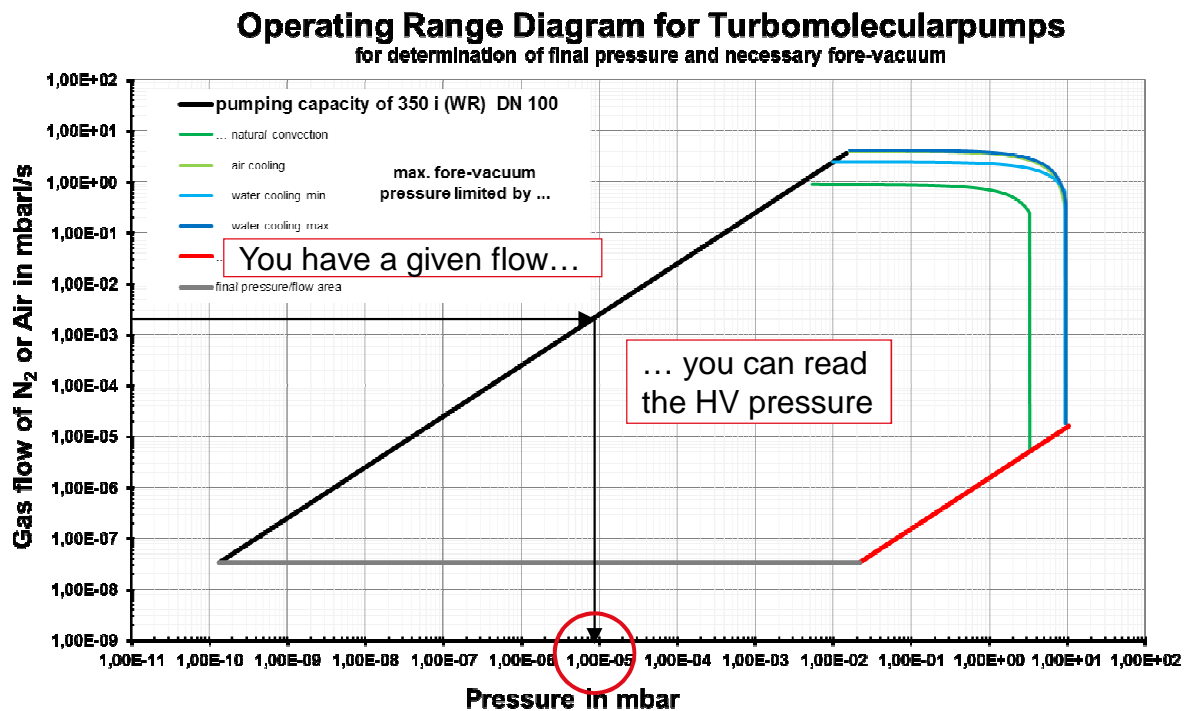
The Qp diagram cannot only show the throughput, but also other useful bits of information!



# The operational diagram: The turbo's "ID card"



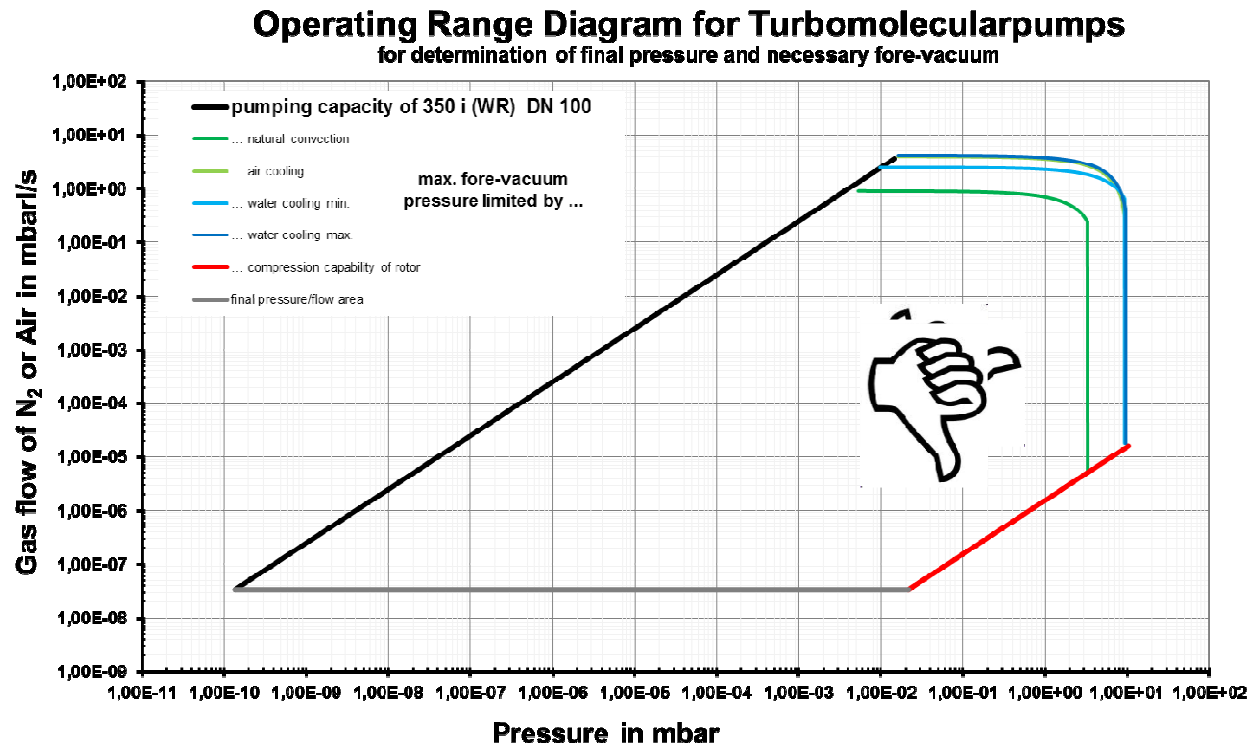
How to read and derive information from the Qp diagram



# The operational diagram: The turbo's "ID card"



You learn if a certain backing pump is a suitable candidate



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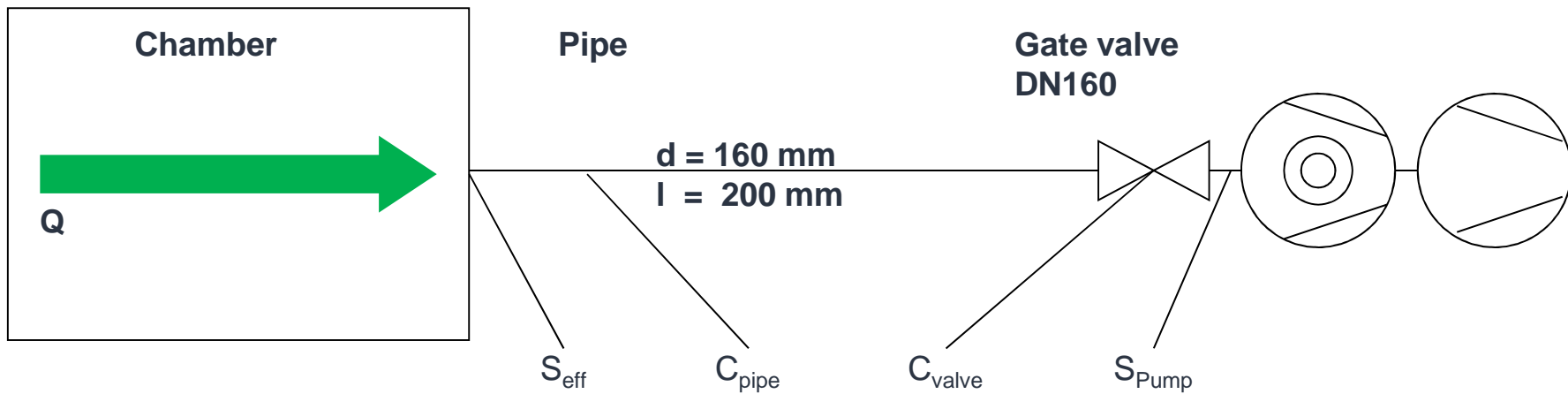


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# Conductance in HV- example calculation



Influence of the suction pipe/Conductance

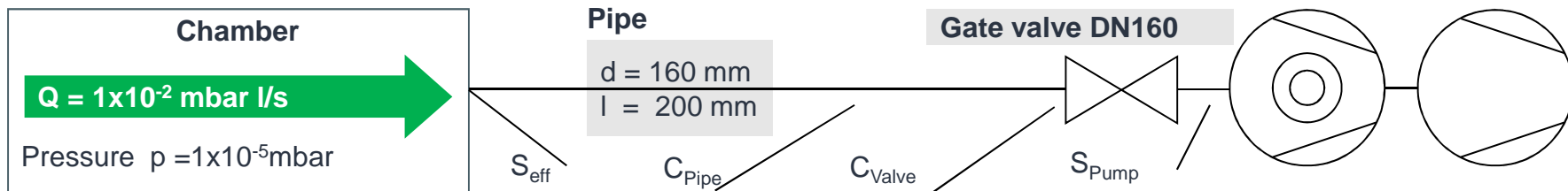


$$\frac{1}{S_{\text{eff}}} = \frac{1}{S_{\text{pump}}} + \frac{1}{C_{\text{Pipe}}} + \frac{1}{C_{\text{Value}}}$$

# Conductance in HV – Example calculation



## Example: Consideration of Conductance (DN160)



$$S_{eff} = \frac{Q(\text{mbar l/s})}{p(\text{mbar})} = \frac{1 \times 10^{-2} \text{ mbar l/s}}{1 \times 10^{-5} \text{ mbar}} = 10^3 [\text{l/s}]$$

Conductance			
Pipe	DN160; 0,2m	$C_{Pipe}$	= 1768 l/s
Gate valve	DN160	$C_{Valve}$	= 6000 l/s

$$\frac{1}{S_{eff}} = \frac{1}{S_{pump}} + \frac{1}{C_{Pipe}} + \frac{1}{C_{Valve}}$$

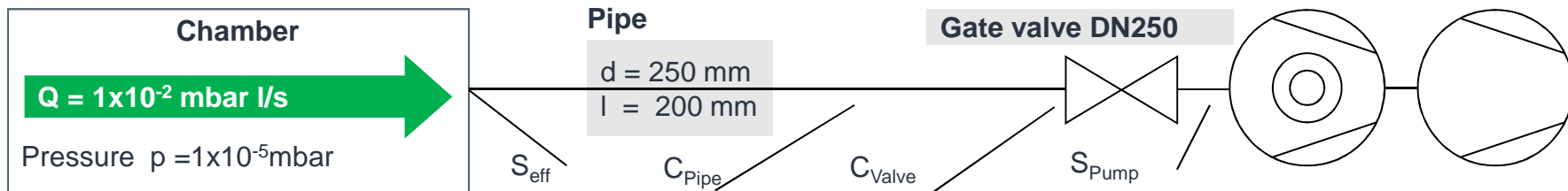
$$S_{pump} = \frac{1}{\frac{1}{S_{eff}} - \frac{1}{C_{pipe}} - \frac{1}{C_{valve}}} = \frac{1}{\frac{1}{1.000} - \frac{1}{1.768} - \frac{1}{6.000}} = 3.735 \text{ l/s}$$



# Conductance in HV – Example calculation



## Example: Consideration of Conductance (DN250)



$$S_{eff} = \frac{Q(\text{mbar l/s})}{p(\text{mbar})} = \frac{1 \times 10^{-2} \text{ mbar l/s}}{1 \times 10^{-5} \text{ mbar}} = 10^3 [\text{l/s}]$$

Conductance			
Pipe	DN250; 0,2m	$C_{Pipe}$	= 5.278 l/s
Gate valve	DN250	$C_{Valve}$	= 22.000 l/s

$$\frac{1}{S_{eff}} = \frac{1}{S_{pump}} + \frac{1}{C_{Pipe}} + \frac{1}{C_{Valve}}$$

$$S_{pump} = \frac{1}{\frac{1}{S_{eff}} - \frac{1}{C_{pipe}} - \frac{1}{C_{valve}}} = \frac{1}{\frac{1}{1.000} - \frac{1}{5.278} - \frac{1}{22.000}} = 1.307 \text{ l/s}$$

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## Summary – Rules for working in HV conditions



### Basic rules to obtain low pressures and fast pump down times

- Minimize surface area inside chamber
- Use materials with low desorption rates
- suitable pre treatment of materials (e.g. electro polishing)
- no internal gaps or trapped volumes
- heating or cooling of chamber surfaces
- reduction of sealings, feedthroughs etc.
- Sufficiently high installed pumping speed with **high connection conductance:**
  - Minimise pipe length
  - Maximise diameters
  - Avoid valves, elbows, reducers etc
  - Use components with smooth (inner surfaces)

Thank you for your attention!

